# Analysis Of Structural and Thermal Characteristics Of Disc Brake With Circular / Oval Shaped Holes For Two Wheeler

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**ABSTRACT-** In this paper study has conducted on oval grooved and circular grooved disc brake rotor of passenger vehicle with full load of capacity. The study is more likely concern of heat and temperature distribution on disc brake rotor. Thermal analysis has been carried through the heat transfer analysis where to predict the worst case scenario and temperature behaviours of disc brake rotor. Further, finite element analysis, ANSYS software has been used in order to identify the temperature distributions and behaviours of disc brake rotor in steady state responses. The results have been compared for better justification. Thus, the results provide better understanding on the thermal characteristic of disc brake rotor and assist the automotive industry in developing optimum and effective disc brake rotor in two wheeler.

KEYWORDS- Disc brake, Circular hole, Oval hole, ANSYS, Temperature distribution.

I.

# INTRODUCTION

### 1.1 ABOUT THE DISC BRAKE

A brake is a device by means of which artificial frictional resistance is applied to moving machine member, in order to stop the motion of a machine. In the process of performing this function, the brakes absorb either kinetic energy of the moving member or the potential energy given up by objects being lowered by hoists, elevators etc. The energy absorbed by brakes is dissipated in the form of heat. This heat is dissipated in the surrounding atmosphere to stop the vehicle, so the brake system should have following requirements: The brakes must be strong enough to stop the vehicle with in a minimum distance in an emergency. The driver must have proper control over the vehicle during braking and vehicle must not skid. The brakes must have well anti fade characteristics i.e. their effectiveness should not decrease with constant prolonged application. The brakes should have well anti wear properties.

Brake technology, just like suspension technology and fuel-system technology, has come a long way in recent years. What began in the '60s as a serious attempt to provide adequate braking for performance cars has ended in an industry where brakes range from supremely adequate to downright phenomenal.

One of the first steps taken to improve braking came in the early '70s when manufacturers, on a widespread scale, switched from drum to disc brakes. Since the Commercial vehicles have been a crucial part of our industrial landscape for hundreds of years. Majority of a vehicle's stopping power is contained in the front wheels, only the front brakes were upgraded to disc during much of this period.

Since then, many manufacturers have adopted four-wheel disc brakes on their high-end and performance models as well as their low-line economy TRUCKS. The development of friction brakes in motor vehicles began with the use of wood and leather as friction material.

# EXPERIMENTAL WORK/ METHODOLOGY

### 2.1 THE DISC BRAKE

The following methodology is adopted in the paper to meet the above mentioned objective.

- Import the geometry of disc brake to ANSYS
- Meshing of the geometry using tetra mesh in ANSYS
- Applying material properties, boundary conditions
- Performing Thermal and Structural response analysis.

### 2.2 MATERIAL PROPERTIES

• Thermal Conductivity =  $52e-3 \text{ W/mm}^{0}\text{K}$ 

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- Specific Heat =  $447 \text{ J/kg}^{0}\text{K}$
- Yield Strength = 275.7 MPa
- Tensile Strength = 413.6 MPa
- Elastic Modulus = 1.9e5 MPa

#### • Poisson's Ratio = 0.27

#### **2.3 ELEMENT DESCRIPTION**

Thermal element: SOLID70 Element DescriptionSOLID70 has a 3-D thermal conduction apability. The element has eight nodes with a single degree of freedom, temperature, at each node. The element is applicable a 3-D, steady-state or transient thermal analysis. The element also can compensate for mass transport heat flow from a constant velocity field.

Structural element: SOLID185 Element DescriptionSOLID185 is used for 3-D modeling of solid structures. It is defined by eight nodes having three degrees of freedom at each node: translations in the nodal x, y, and z directions. The element has plasticity, hyper elasticity, stress stiffening, creep, large deflection, and large strain capabilities. It also has mixed formulation capability for simulating deformations of nearly incompressible elasto- plastic materials, and fully incompressible hyper elastic materials.



Figure 2.2 Solid 185 geometry

#### 2.4 FINITE ELEMENT ANALYSIS

The analysis is done in ANSYS by taking the Brake Efficiency of 30% and hence the distributions of braking torque between the front and rear axle is 70:30

### Thus Load Calculations:

Load = 520103.52 N Pressure = load/Area=1 MPa

1 lessure = 10 au/Area = 1 w

Temperature =  $100^{\circ}$ C

Convection =  $300E-6 \text{ W/mm}^2 \text{ }^0\text{K}$ 

Bulk Temperature =  $50^{\circ}$ C

The procedure for doing analysis is as follows:

First thermal analysis is performed by importing geometric model from ANSYS and applying SOLID70 as element, and then the model is meshed. The temperature of  $100^{0}$ C, Convection of 300E-6 W/mm<sup>2</sup> <sup>0</sup>K, Bulk Temperature 50<sup>0</sup>C are applied as boundary conditions and then is solved. The nodal temperatures are plotted. For doing structural analysis the element is switched from thermal to structural analysis. Selecting SOLID185

element assigning material properties: young's modulus as 1.9e5 N/mm<sup>2</sup>, poisson's ratio as 0.27, and thermal expansion as 10.8e-6. Applying displacement constraints and pressure as 1Mpa. Then browse .RthFile from thermal Analysis and solved to get the von-misses stresses.

### 2.5 GEOMETRIC MODELING

ANSYS itself is used for modeling of disc brake rotor with oval and circular hole. Then the model is imported in to ANSYS for doing analysis.



Figure 2.3: Geometric model of disc brake with circular hole



Figure 2.4: Geometric model of disc brake with oval hole

## III. RESULTS AND DISCUSSION



Figure 3.1.Nodal temperature of disk brake with circular hole





Figure 3.2.Deformation of disk brake with circular hole Figure 3.2.Shows Total Deformation of Disc Brake. Deformation is 0.144402 mm



Figure 3.3. Von-mises stress of disk brake with circular hole

Figure 3.3. Shows minimum and maximum Vonmises Stress distribution with Circular hole. Minimum value is 0.030533 MPa and Maximum value is 370.653 MPa.



Figure 3.4. Body temperature of disc plate with circular hole

### 3.2. RESULTS OF DISC PLATE WITH OVAL HOLE:



Figure 3.5.Nodal temperature of disk brake with oval hole

Figure 3.5.Shows Minimum and Maximum Temperature distribution with oval hole. Minimum value is 29.9859°C and Maximum value is 70°C.



Figure 3.6.Deformation of disk brake with oval hole Figure 3.6.Shows Total Deformation of Disc Brake. Deformation is 0.14252 mm.



Figure 3.7. Von-mises stress of disk brake with oval hole

Figure 3.7.Shows minimum and maximum Vonmises Stress distribution with Oval hole. Minimum value is 0.001445 MPa and Maximum value is 191.521 MPa.



Figure 3.8. Body temperature of disc plate with oval hole

Table 5.1.1 (oddi temperature, deformation and von mises suesses in the brake				
Parameters	Disc Plate With Circular	Disc Plate With Oval Holed		
	Holed			
Nodal Temperature (°C)	76	70		
Deformation (mm)	0.144402	0.14252		
Von-mises stress(MPa)	370.653	191.521		

Table 3.1.Nodal temperature, deformation and von-mises stresses in the brake

### 3.3 EXPERIMENTAL RESULTS:

Table 3.1 Shows the temperature distribution of disc plate during heating.

TIME	CIRCULAR HOLED DISC PL	OVAL HOLED DISC PLATE
(191111.)	IEWIPERATURE (C)	IEMPERATURE (C)
0	35.0	35.0
5	43.1	41.4
10	48.7	46.9
15	57.6	55.2
20	63.1	61.0
25	74.0	69.5
30	79.9	73.2

Table 3.1 Temperature Rise during heating



Figure 3.9. Graph for temperature distribution

The above graph shows the temperature rise of both disc plates that clarify the disc plate with oval hole gives better result.



#### Figure 3.10.Experimental setup

The above experimental setup is used for investigating the temperature generation and temperature distribution of both circular and oval shaped holes disc plates. The values are noted and tabulated below. Table 3.2 Experiment results of both disc plates

TIME (Min.)	CIRCULAR HOLED DISC PI TEMPERATURE ( <sup>0</sup> C)	TIME (Min.)	OVAL HOLED DISC P TEMPERATURE ( <sup>0</sup>
0	35.5 (Heat Gent.)	0	35.5 (Heat Gent.)
5	35.9	5	35.8
10	38.2	10	37.4
15	40.4	15	38.9
20	42.1	20	40.0
25	42.1	25	40.2
26	42.7 (Peak Temp.)	29	39.8 (Peak Temp.)
30	42.1 (Heat Disp.)	30	38.8 (Heat Disp.)
35	41.7	35	37.6
40	40.8	40	36.5
45	40.5	45	35.9
50	39.4	50	35.4
55	39.2	55	35.0
60	38.4	60	34.8

Table.3.2. Shows the temperature generation and temperature dissipation of both circular and oval holed disc plates. From this it concluded the oval holed disc plate having better heat distribution and dissipation.

### IV. CONCLUSION

In this paper, the Thermal and Structural analysis of disk brakes with circular and oval Hole has been performed. The effect of hole with circular and oval shapes is examined. From results we can say that all the values obtained from the Thermal analysis are less than their allowable values. Hence the brake disk design is safe based on the temperature criteria. Comparing the circular and oval holed disc plates, the thermal results obtained from analysis to conclude which is the best possible combination for the present application.

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